

# *Numerical Solution of Ordinary Differential Equations*

Most fundamental laws of Science are based on models that explain variations in physical properties and states of systems described by **differential equations**.

Several examples of laws appear in C&C PT 7.1 and are applied in Ch. 28 --

- Newton's 2nd law:  $\frac{dv}{dt} = \frac{F}{m}$  velocity(v), force (F), and mass (m)
- Fourier's heat law:  $q = -k' \frac{dT}{dx}$  heat flux (q), temperature (T), and thermal conductivity (k')
- Fick's diffusion law  $j = -D \frac{dc}{dx}$  mass flux (j), concentration (c) and diffusion coefficient (D)
- Faraday's law:  $\Delta V_i = L \frac{di}{dt}$  voltage drop ( $\Delta V$ ), inductance (L) and current (i)

## **ODE's** Ordinary Differential Equations

Only **one** independent variable, i.e.,  $\mathbf{x}$  as in  $y(\mathbf{x})$ :

$$\frac{d^2 y}{dx^2} + \omega^2 y = g(x) \quad \text{1-dimensional problem in space } x$$

$$\frac{dy}{dt} = f(y, t) \quad \text{time-dynamics problem}$$

## **PDE's** Partial Differential Equations

More than one independent variable, i.e.,  $\mathbf{x}$  and  $\mathbf{y}$  as in  $T(\mathbf{x}, \mathbf{y})$ :

$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = 0 \quad \text{Laplace Equation}$$

$$\frac{\partial^2 u}{\partial t^2} = c^2 \frac{\partial^2 u}{\partial x^2} = 0 \quad \text{Wave Equation } u = u(\mathbf{x}, t)$$

## ***Auxiliary Conditions***

Because we are integrating an indefinite integral, we need additional information to obtain a unique solution. Note that an  $n^{\text{th}}$  order equations generally requires  $n$  auxiliary conditions.

- **Initial Value Problem (IV)**

Information at a single value of the independent variable, typically at the beginning of the interval:

$$y(0) = y_0$$

- **Boundary Value Problem (BV)**

Information at more than one value of the independent variable, typically at both ends of the interval:

$$y(0) = y_0 \text{ and } y(1) = y_f$$

## **I. Single-Step Methods for IV Problems (C&C Ch. 25)**

- a. Euler
- b. Heun and Midpoint/Improved Polygon
- c. General Runge-Kutta
- d. Adaptive step-size control

## **II. Stiff ODEs (C&C 26.1)**

## **III. Multi-Step Methods for IV Problems (C&C 26.2)**

- a. Non-Self-Starting Heun
- b. Newton-Cotes
- c. Adams

## **IV. Boundary Value (BV) Problems (C&C Ch. 27.1 & 27.2)**

- a. Solution Methods
  1. Shooting Method
  2. Finite Difference Method
- b. Eigenvalue Problems  $\{ [A] - \lambda [I] \} \{x\} = \{0\}$

## **One-Step Methods for IV Problems**

Consider the generic first-order initial value (IV) ODE problem:

$$\frac{dy}{dx} = f(x, y) \quad \text{where } (x_0, y_0) \text{ are given}$$

and we wish to find  $y = y(x)$

The independent variable  $\mathbf{x}$   
may be space  $x$ ,  
or time  $t$ .

## *ODE's: One-step methods*

**We can solve higher-order IV ODE's  
by transforming to a set of 1st-order ODE's,**

$$\frac{d^2y}{dx^2} + \frac{dy}{dx} + 5y = 0$$

$$\text{Let } z = \frac{dy}{dx} \text{ \& substitute } \rightarrow \frac{dz}{dx} + z + 5y = 0$$

Now solve a SYSTEM of two linear, first order ordinary differential equations:

$$\frac{dy}{dx} = z \quad \text{and} \quad \frac{dz}{dx} = -z - 5y$$

## *ODE's: first order IV problem - One-step methods*

$$\frac{dy}{dx} = f(x, y) \quad \text{where } (x_0, y_0) \text{ are given and we wish to find } y = y(x).$$

**The basic approach to numerical solution is stepwise:**

Start with  $(x_0, y_0) \Rightarrow (x_1, y_1) \Rightarrow (x_2, y_2) \Rightarrow \text{etc.}$

Next Value = Previous Value + *slope* × step size

$$y_{i+1} = y_i + \phi_i \times h$$

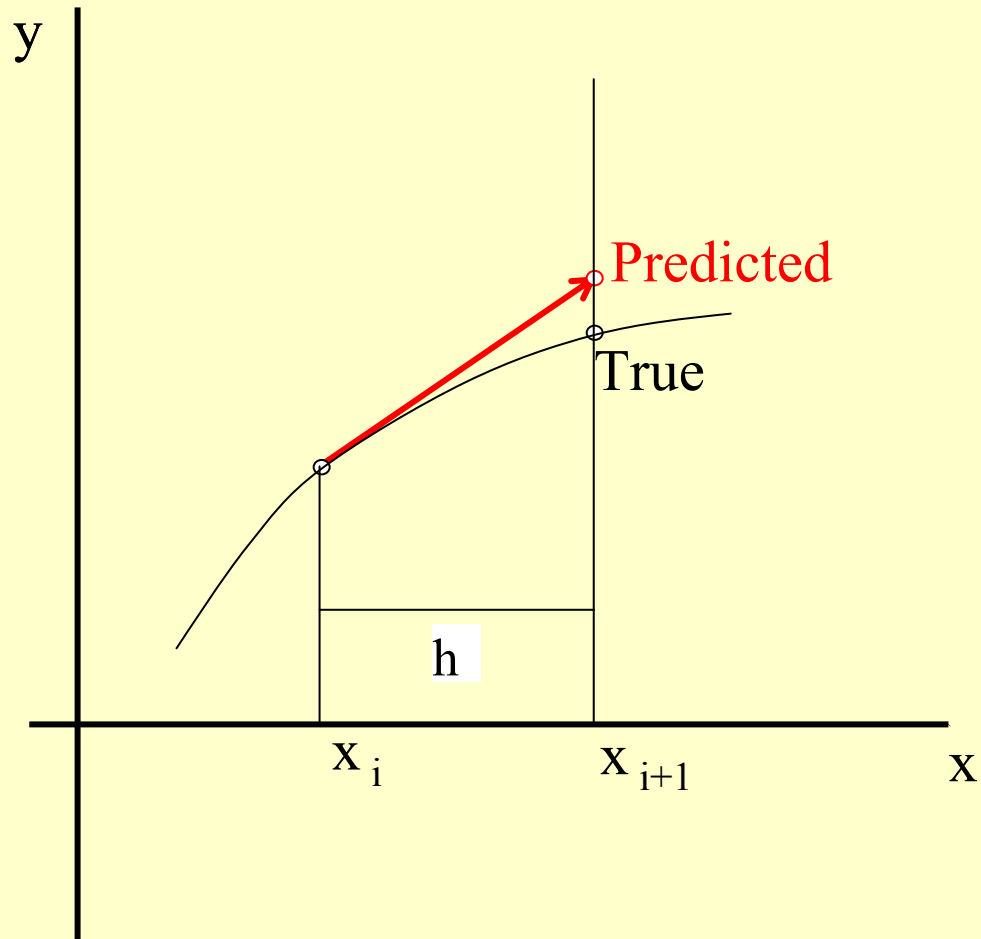
$$h = x_{i+1} - x_i = \text{step size}$$

Key to the various one-step methods is how the *slope* is obtained.

This *slope* represents a weighted average of the slope over the entire interval and may not be the tangent at  $(x_i, y_i)$



# Using an estimate of the slope to guess $f(x_{i+1})$



## *Euler's Method*

Given  $(x_i, y_i)$ , need to determine  $(x_{i+1}, y_{i+1})$

$$x_{i+1} = x_i + h$$

$$y_{i+1} = y_i + \phi_i h$$

Estimate the *slope* as  $\phi_i = f(x_i, y_i)$

$$y_{i+1} = y_i + f(x_i, y_i) h$$

The slope at the **beginning** of the step is applied across the entire interval

## **Analysis Local Error for the Euler Method**

Taylor series of true solution:  $y_{i+1} = y_i + y_i' h + y_i'' h^2/2 + \dots$

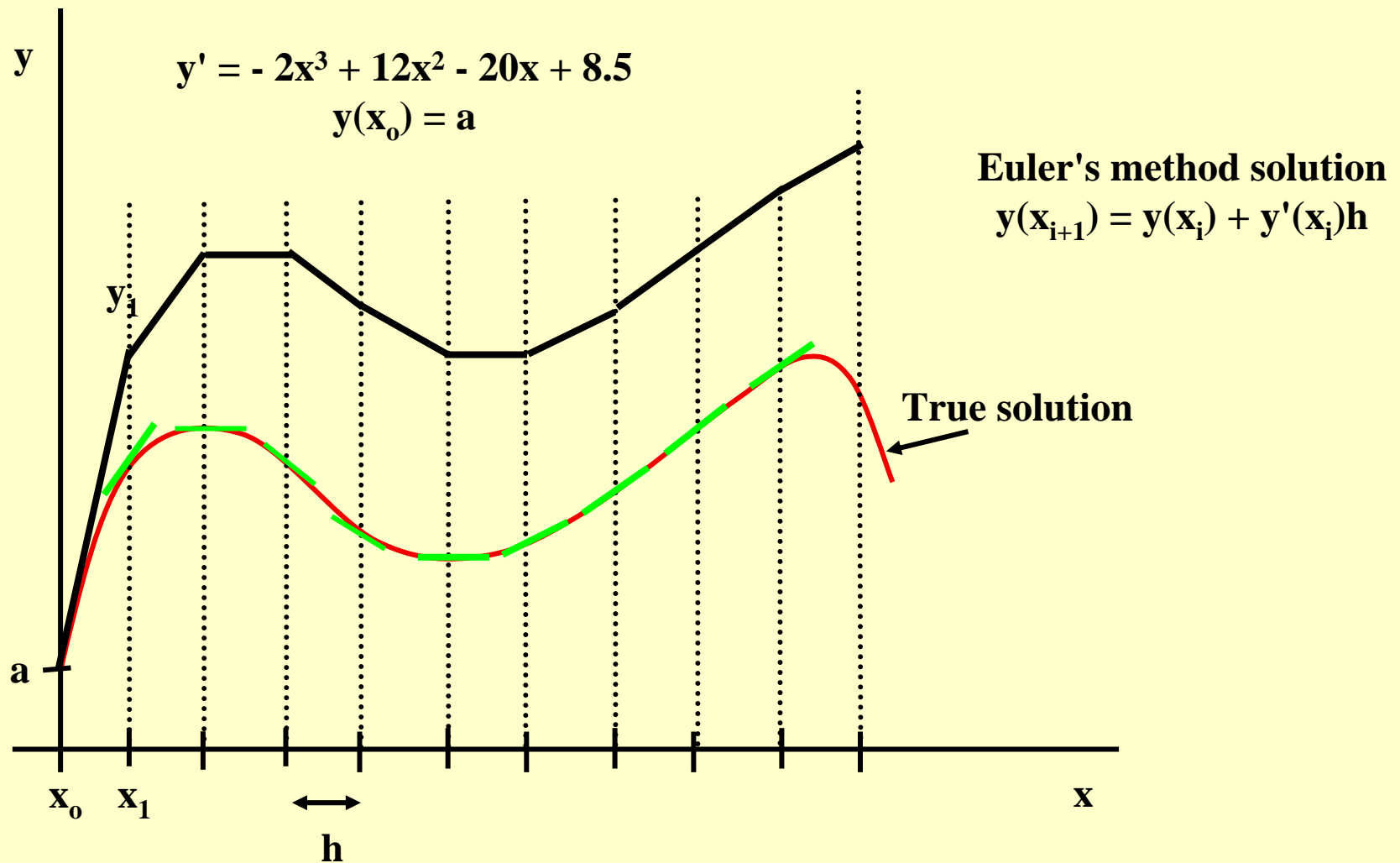
where  $y_i' = f(x_i, y_i) = f_i$

Euler rule:  $\hat{y}_{i+1} = y_i + f_i h$

Truncation error:  $E_a = y_{i+1} - \hat{y}_{i+1} = y_i'' \frac{h^2}{2} = O(h^2)$   
*(local)*

Global Error:  $E_a = G(n, h) O(h^2) = O(h)$

# Euler's Method: Graphic example



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## *Error Analysis*

1. Truncation Error (Truncating Taylor Series)  
[large step size → large errors]
2. Rounding Error (Machine precision)  
[very small step size → roundoff errors]

### *Two kinds of Truncation Error:*

***Local*** – error within one step due to application of the numerical method

***Propagation*** – error due to previous local errors.

*Global Truncation Error* = Local + Propagation

Generally if the local truncation error is  $O(h^{n+1})$  then, as with numerical quadrature formulas, the global truncation error is  $O(h^n)$ . (Proof is more difficult.)

### *General Error Notes:*

1. For stable systems, error is reduced by decreasing  $h$
2. If method is  $O(h^n)$  globally, then it is exact for  $(n-1)^{\text{th}}$  order polynomial in  $x$

## *Improved One-Step Methods*

### *Heun's Method (simple predictor-corrector)*

Predict:  $y_{i+1}^0 = y_i + f(x_i, y_i)h$

Estimate the avg. slope as:  $\bar{f} = \frac{f(x_i, y_i) + f(x_{i+1}, y_{i+1}^0)}{2}$

Correct:  $x_{i+1} = x_i + h \quad y_{i+1} = y_i + \bar{f}h$

**Notes:** 1. Local Error  $O(h^3)$  and Global Error  $O(h^2)$

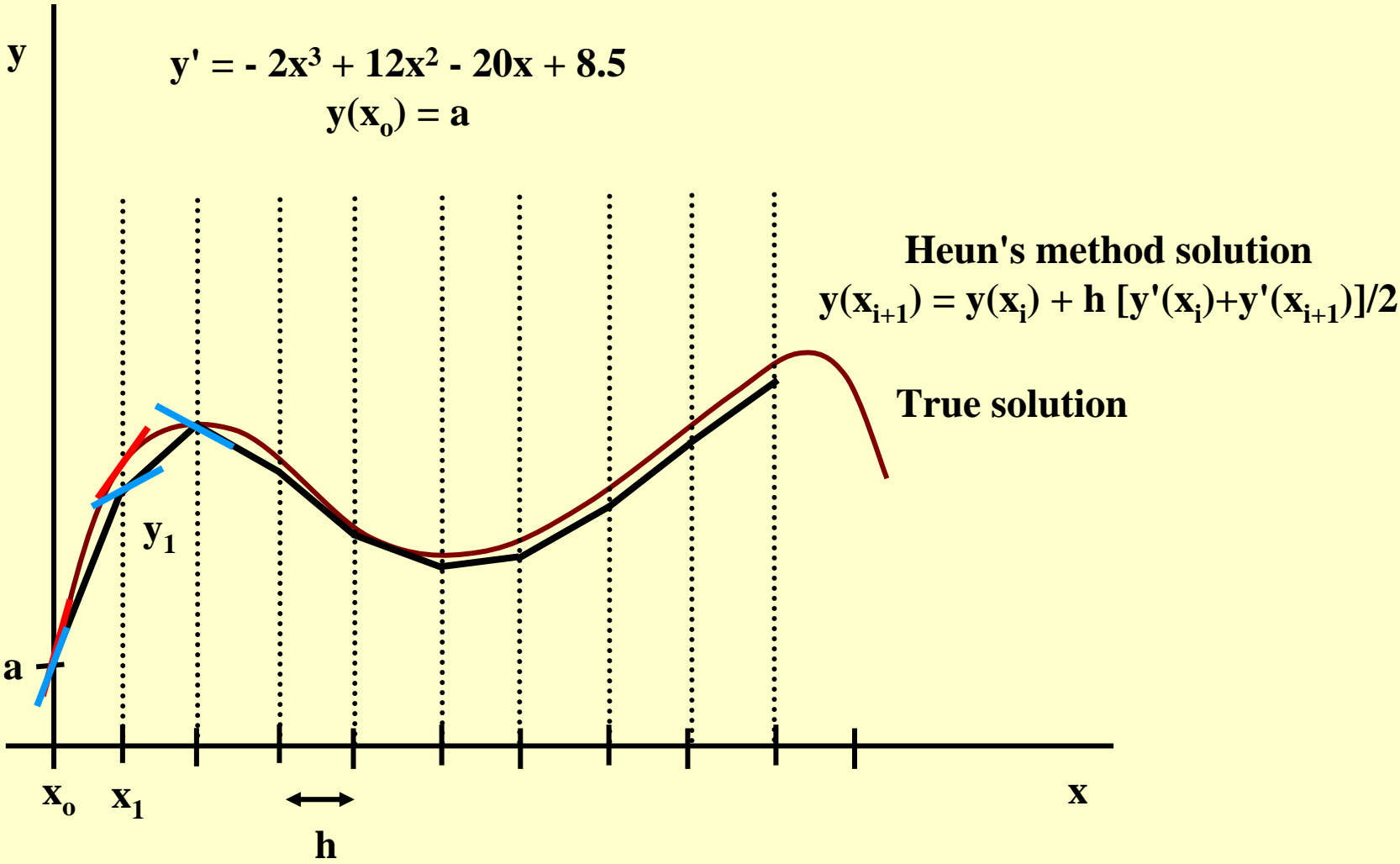
2. Corrector step can be iterated using  $\epsilon_a$  as stop criterion

3. If derivative is only  $f(x)$  and not  $f(x, y)$ , then the predictor has no effect and

$$y_{i+1} = y_i + \frac{f(x_i) + f(x_{i+1})}{2}h$$

which is the Trapezoid Rule.

Euler's Method: Graphic example



## *Midpoint Method (Improved Polygon, Modified Euler)*

- Predict  $y_{i+1/2}$  with a half step:

$$y_{i+1/2} = y_i + f(x_i, y_i) \frac{h}{2}$$

- Estimate the slope:

$$slope = f(x_{i+1/2}, y_{i+1/2}) \quad \text{with} \quad x_{i+1/2} = x_i + \frac{h}{2}$$

- Correct  $y_{i+1}$ :  $y_{i+1} = y_i + f(x_{i+1/2}, y_{i+1/2}) h$

**Note:** Local Error  $O(h^3)$  and Global Error  $O(h^2)$   
Error same order as Heun.

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## *General Runge-Kutta (RK) Methods*

Employing single-step

$$y_{i+1} = y_i + \phi(x_i, y_i, h) h$$

Increment function or slope,  $\phi(x_i, y_i, h)$ , is a weighted average:

$$\phi = a_1 k_{1i} + a_2 k_{2i} + \dots + a_j k_{ji} + \dots + a_n k_{ni} \quad (n^{\text{th}} \text{ order method})$$

where:  $a_j =$  weighting factors (that sum to unity)

$k_{ji} =$  slope at point  $x_{ji}$  such that  $x_i < x_{ji} < x_{i+1}$

$$k_{1i} = f(x_i, y_i)$$

$$k_{2i} = f(x_i + p_1 h, y_i + q_{11} k_{1i} h)$$

$$k_{3i} = f(x_i + p_2 h, y_i + q_{21} k_{1i} h + q_{22} k_{2i} h)$$

•  
•  
•

$$k_{ni} = f(x_i + p_{n-1} h, y_i + q_{n-1,1} k_{1i} h + q_{n-1,2} k_{2i} h + \dots + q_{n-1,n-1} k_{n-1,i} h)$$



## ***Runge-Kutta (RK) Methods***

**First-order Runge-Kutta Method:** Euler's Method

$$y_{i+1} = y_i + k_1 h \quad k_1 = f(x_i, y_i) \quad a_1=1$$

Has a global truncation error,  $O(h)$

**Second-order Runge-Kutta Methods:**

Assume  $a_2 = 1/2$ ; Heun's w/ Single Corrector: no iteration

$$y_{i+1} = y_i + \left(\frac{1}{2}k_1 + \frac{1}{2}k_2\right)h \quad \begin{array}{l} k_1 = f(x_i, y_i) \\ k_2 = f(x_i + h, y_i + hk_1) \end{array}$$

Assume  $a_2 = 1$ ; Midpoint (Improved Polygon)

$$y_{i+1} = y_i + k_2 h \quad k_2 = f\left(x_i + \frac{1}{2}h, y_i + \frac{1}{2}hk_1\right) \quad \text{same } k_1$$

Assume  $a_2 = 2/3$ ; Ralston's Method (minimum bound on  $E_t$ )

$$y_{i+1} = y_i + \left(\frac{1}{3}k_1 + \frac{2}{3}k_2\right)h \quad k_2 = f\left(x_i + \frac{3}{4}h, y_i + \frac{3}{4}hk_1\right) \quad \text{same } k_1$$

All have global truncation error,  $O(h^2)$

## *Classical Fourth-order Runge-Kutta Method*

### **Classical Fourth-order Runge-Kutta Method**

Employing single-step:

$$y_{i+1} = y_i + (1/6) [ k_1 + 2k_2 + 2k_3 + k_4 ] h$$

where	$k_1 = f(x_i, y_i)$	Euler step
	$k_2 = f(x_i + h/2, y_i + k_1 h/2)$	Midpoint
	$k_3 = f(x_i + h/2, y_i + k_2 h/2)$	Better midpoint
	$k_4 = f(x_i + h, y_i + k_3 h)$	Full step

1. Global truncation error,  $O(h^4)$
2. If the derivative,  $f$ , is a function of only  $x$ , this reduces to Simpson's 1/3 Rule.

$$y_{i+1} = y_i + h (k_1 + 2k_2 + 2k_3 + k_4)/6 \quad \text{where } k_2 = k_3$$

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## *Examples of Frequently Encountered, Simple ODE's*

### 1. **Exponential growth**

unconstrained growth of biological organisms,  
positive feedback electrical systems, and  
chemical reactions generating their own catalyst)

$$\frac{dy}{dt} = \lambda y \quad \text{with solution} \quad y(t) = y_0 e^{\lambda t}$$

### 2. **Exponential decay**

discharge of a capacitor,  
decomposition of material in a river,  
wash-out of chemicals in a reactor, and  
radioactive decay

$$\frac{dy}{dt} = -\lambda y \quad \text{with solution} \quad y(t) = y_0 e^{-\lambda t}$$

## *Classical Fourth-order Runge-Kutta Method -- Example*

### **Numerical Solution of the simple differential equation**

$y' = + 2.77259 y$  with  $y(0) = 1.00$ ; Solution is  $y = \exp(+2.773 x) = 16^x$

Step sizes vary so that all methods use the same number of functions evaluations to progress from  $x = 0$  to  $x = 1$ .

#### **4th-order**

<b>x</b>	<b>Exact Solution</b>	<b>Euler</b>	<b>Heun w/o iter</b>	<b>Runge-Kutta</b>	<b>h * ki for R-K</b>
0.000	1.000	1.000	1.000	1.000	
0.125	1.414	1.347			1.386
0.250	2.000	1.813	1.933		2.347
0.375	2.828	2.442			3.013
0.500	4.000	3.288	3.738	3.945	5.564
0.625	5.657	4.427			5.469
0.750	8.000	5.962	7.227		9.260
0.875	11.31	8.028			11.89
1.000	16.00	10.81	13.97	15.56	21.95
	<b><i>h =</i></b>	<b><i>0.125</i></b>	<b><i>0.25</i></b>	<b><i>0.5</i></b>	

## *Classical Fourth-order Runge-Kutta Method – Example (cont.)*

<b>x</b>	<b>Exact Solution</b>	<b>Euler</b>	<b>Heun w/o iter</b>	<b>4th-order Runge- Kutta</b>	<b>h * ki for R-K</b>
0.0000	1.0000	1.0000	1.0000		
0.0625	1.1892	1.1733			0.6931
0.1250	1.4142	1.3766	1.4066		0.9334
0.1875	1.6818	1.6151			1.0166
0.2500	2.0000	1.8950	1.9786	1.9985	1.3978
0.3125	2.3784	2.2234			1.3853
0.3750	2.8284	2.6087	2.7832		1.8653
0.4375	3.3636	3.0608			2.0317
0.5000	4.0000	3.5911	3.9149	3.9940	2.7935
0.5625	4.7568	4.2134			2.7684
0.6250	5.6569	4.9436	5.5068		3.7279
0.6875	6.7272	5.8002			4.0604
0.7500	8.0000	6.8053	7.7460	7.9820	5.5829
0.8125	9.5137	7.9846			5.5327
0.8750	11.314	9.3683	10.8958		7.4502
0.9375	13.454	10.992			8.1147
1.0000	16.000	12.896	15.326	15.952	11.157

***h =***

***0.0625***

***0.125***

***0.25***

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## *Classical Fourth-order Runge-Kutta Method*

In each case, all three RK methods used the same number of function evaluations to move from 0.00 to 1.00.

Which was able to provide the more accurate estimate of  $y(1)$ ?

## **Higher-Order ODEs and Systems of Equations (C&C 25.4, p.711)**

- An  $n^{\text{th}}$  order ODE can be converted into a system of  $n$ , coupled 1st-order ODEs.
- Systems of first order ODEs are solved just as one solves a single ODE.

Consider the 4th-order ODE:

$$f(x) = y'''' + a(x) y''' + b(x) y'' + c(x) y' + d(x) y$$

Let:  $y''' = v_3$ ;  $y'' = v_2$ ; and  $y' = v_1$

Write this 4<sup>th</sup> order ODE as a system of four coupled 1<sup>st</sup> order ODEs:

$$\frac{d}{dx} \begin{pmatrix} y \\ v_1 \\ v_2 \\ v_3 \end{pmatrix} = \begin{pmatrix} v_1 \\ v_2 \\ v_3 \\ f(x) - a(x)v_3 - b(x)v_2 - c(x)v_1 - d(x)y \end{pmatrix}$$

## Higher-Order ODEs and Systems of Equations (C&C 25.4, P. 711)

$$\begin{pmatrix} dy/dx \\ dv_1/dx \\ dv_2/dx \\ dv_3/dx \end{pmatrix} = \begin{pmatrix} v_1 \\ v_2 \\ v_3 \\ f(x) - a(x)v_3 - b(x)v_2 - c(x)v_1 - d(x)y \end{pmatrix}$$

Given the initial conditions @  $x = 0$ :

$$y(0); v_1 = y'(0); v_2 = y''(0); v_3 = y'''(0),$$

a numerical scheme can be used to integrate this system forward in time.



## *Example: Harmonically Driven Oscillator*

**Example: Single-degree-of-freedom, undamped, harmonically driven oscillator (see also C&C 28.4, p. 797)**

ODE is 2nd order:

$$m \frac{d^2 x}{dt^2} + kx = P(t) \quad \text{or} \quad \frac{d^2 x}{dt^2} + \omega^2 x = \frac{P(t)}{m}$$

$m$  = mass

$k$  = spring constant

$P(t)$  = forcing function =  $P \sin(\gamma t)$   
i.e., harmonically driven

$\omega = \sqrt{\frac{k}{m}}$  = circular frequency

$T = \frac{2\pi}{\omega}$  = natural period

## *Example: Harmonically Driven Oscillator*

$$\frac{d^2x}{dt^2} + \omega^2 x = \frac{P(t)}{m}$$

Initial conditions:  $x(0) = 0$  and  $\frac{dx}{dt}(0) = 0$

Analytical solution:

$$\begin{aligned} x(t) &= \frac{-P\gamma}{\omega m(\omega^2 - \gamma^2)} \sin(\omega t) + \frac{P}{m(\omega^2 - \gamma^2)} \sin(\gamma t) \\ &= (\text{homogeneous soln.}) + (\text{particular soln.}) \end{aligned}$$

## *Example: Harmonically Driven Oscillator*

$$\frac{d^2x}{dt^2} + \omega^2 x = \frac{P(t)}{m}$$

Recast the 2<sup>nd</sup>-order ODE as two 1<sup>st</sup>-order ODEs):

$$\frac{dx}{dt} = v = f(t, x, v) \quad \text{with } x(0) = 0$$

$$\frac{dv}{dt} = \frac{P \sin(\gamma t)}{m} - \omega^2 x = g(t, x, v) \quad \text{with } v(0) = 0$$

## Example: Harmonically Driven Oscillator – Euler Method

$$\frac{d^2x}{dt^2} + \omega^2 x = \frac{P(t)}{m} \quad \begin{array}{l} \longrightarrow \\ \searrow \end{array} \quad \begin{array}{l} \frac{dx}{dt} = v = f(t, x, v) \\ \frac{dv}{dt} = \frac{P \sin(\gamma t)}{m} - \omega^2 x = g(t, x, v) \end{array}$$

We can solve these two 1st-order ODE's sequentially.

Let  $m = 0.25$ ,  $k = 4\pi^2$ ,  $P = 20$ ,  $\gamma = 8$ ,

Thus  $g = 80 \sin(8t) - 16\pi^2 x$ . Solve from  $t = 0$  until  $t = 1$ .

For example, by the Euler method, with  $h = 0.01$ :

<b>t</b>	<b>x</b>	<b>v</b>
$t_0 = 0$	$x_0 = 0$	$v_0 = 0$
$t_1 = t_0 + h = 0.01$	$x_1 = x_0 + f(t_0, x_0, v_0)h = 0$	$v_1 = v_0 + g(t_0, x_0, v_0)h = 6.39$
$t_2 = t_1 + h = 0.02$	$x_2 = x_1 + f(t_1, x_1, v_1)h = 0.0639$	$v_2 = v_1 + g(t_1, x_1, v_1)h = 12.75$
...	...	...
$t_n = t_{n-1} + h$	$x_n = x_{n-1} + f(t_{n-1}, x_{n-1}, v_{n-1})h$	$v_n = v_{n-1} + g(t_{n-1}, x_{n-1}, v_{n-1})h$

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## Adaptive Step-size Control (C&C 25.5, p. 710)

**Goal:** with little additional effort estimate (bound) magnitude of local truncation error at each step so that step size can be reduced/increased if local error increases/decreases.

1. Repeat analysis at each time step with step length  $h$  and  $h/2$ . Compare results to estimate local error. (C&C 25.5.1) Use Richardson extrapolation to obtain higher order result.
2. Use a matched pair of Runge-Kutta formulas of order  $r$  and  $r+1$  which use common values of  $k_j$ , and yield estimate or bound local truncation error.

C&C 25.5.2 discuss 4th-5th-order Runge-Kutta-Fehlberg pair.

## A 2<sup>nd</sup> – 3<sup>rd</sup> order RK pair yielding local truncation error estimate:

2nd-order Midpoint Method  $O(h^2)$ :

$$\begin{aligned}k_1 &= f(x_i, y_i); & k_2 &= f(x_i + 1/2h, y_i + 1/2hk_1) \\ y_{i+1} &= y_i + k_2 h\end{aligned}$$

Third-order RK due to Kutta  $O(h^3)$  (C&C 25.3.2)

$$\begin{aligned}k_3 &= f(x_i + h, y_i + h(2k_2 - k_1)) \\ y_{i+1}^* &= y_i + h(k_1 + 4k_2 + k_3)\end{aligned}$$

Estimate of truncation error for midpoint formula is

$$E_i = y_{i+1} - y_{i+1}^* = -h(k_1 - 2k_2 + k_3)$$

This is a central difference estimate of (const.)  $h^3 y'''$   
which describes the local truncation error for midpoint method.

Use more accurate values  $y_{i+1}^*$ , but  $E_i$  provides estimate of the local error that can be used for step-size control.

## Stiff Differential Equations (C&C 26.1, p. 719)

A *stiff* system of ODE's is one involving rapidly changing components together with slowly changing ones. In many cases, the rapidly varying components die away quickly, after which the solution is dominated by the slow ones.

Even simple first-order ODE's can be stiff. C&C gives the example:

$$\frac{dy}{dt} = -1000y + 3000 - 2000e^{-t}$$

With initial conditions  $y(0) = 0$  and with solution

$$y(t) = 3 - 0.998e^{-1000t} - 2.002e^{-t}$$

Fast term

Slow term

## Stiff Differential Equations

In the solution

$$y(t) = 3 - 0.998e^{-1000t} - 2.002e^{-t}$$

the middle term damps out very quickly and after it does, the slow solution closely follows the path:

$$y(t) \approx 3 - 2e^{-t}$$

We would need a very small time step  $h = \Delta t$  to capture the behavior of the rapid transient and to preserve a stable and accurate solution, and this would then make it very laborious to compute the slowly evolving solution.

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## Stiff Differential Equations

A **stability analysis** of this equation provides important insights. For a solution to be *stable* means that errors at any stage of computation are not amplified but are attenuated as computations proceed. To analyze for stability, we consider the homogeneous part of the ODE

$$\frac{dy}{dt} = -ay \quad \text{with solution} \quad y(t) = y_0 e^{-at}$$

Euler's method yields

$$y_{i+1} = y_i + \frac{dy_i}{dt} h = y_i - ay_i h = y_i (1 - ah)$$

Assume some small error exists in the initial condition  $y_0$  or in an early stage of the solution. Then we see that after  $n$  steps,

$$y_n = y_0 (1 - ah)^n \quad \text{Bounded iff } |(1-ah)| < 1, \text{ i.e., } h < 2/a$$

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## Stiff Differential Equations

Euler's method is known as an *explicit* method because the derivative is taken at the known point  $i$ . An alternative is to use an *implicit* approach, in which the derivative is evaluated at a future step  $i+1$ . The simplest method of this type is the backward Euler method that yields

$$y_{i+1} = y_i + \frac{dy_{i+1}}{dt} h = y_i - ay_{i+1}h \quad \Rightarrow \quad y_{i+1} = \frac{y_i}{1 + ah}$$

Because  $1/(1+ah)$  will remain bounded for any (positive) value of  $h$ , this method is said to be *unconditionally stable*.

Implicit methods always entail more effort than explicit methods, especially for nonlinear equations or sets of ODE's, so the stability is gained at a price. Moreover, accuracy still governs the step size.

# Systems of Stiff Differential Equations

## What to do?

Need to be aware of the potential for encountering stiff ODE's.

Use special codes for solving stiff differential equations which are generally implicit multistep methods.

For example, MATLAB has some methods specifically designed to solve stiff ODE's, e.g., ode23S.

## Predictor-Corrector Multi-Step Methods for ODE's

- Utilize valuable information at previous points.
- For increased accuracy, use a predictor that has truncation of same order as corrector. In addition iterate corrector to minimize truncation error and improve stability.

### Non-Self Starting (NSS) Heun Method

$$O(h^3)\text{-Predictor} - y_{i+1}^0 = y_{i-1}^m + f(x_i, y_i^m) 2h$$

Higher order predictor than the self-starting Heun.

This is an open integration formula (midpoint).

$$O(h^3)\text{-Corrector} - y_{i+1}^j = y_i^m + \frac{f(x_i, y_i^m) + f(x_{i+1}, y_{i+1}^{j-1})}{2} h$$

- Notes:
1. Corrector is applied iteratively for  $j=1, \dots, m$
  2.  $y_i^m$  is fixed (from previous step iterations)
  3. This is a closed integration formula (Trapezoid)

## Multi-step methods

Two general schemes solving ODE's:

### 1. *Newton-Cotes Formulas*

$$y_{i+1} = \begin{cases} y_{i-n} + h \sum_{k=0}^{n-1} c_k f_{i-k} + O(h^{n+1}) & \text{open predictor} \\ y_{i-n+1} + h \sum_{k=0}^{n-1} \bar{c}_k f_{i-k+1} + O(h^{n+1}) & \text{closed corrector} \end{cases}$$

with  $c_k$  from Table 21.4, and  $\bar{c}_k$  from Table 21.2 of C&C.

### 2. *Adams Formulas* (Generally more stable)

$$y_{i+1} = y_i + \begin{cases} h \sum_{k=0}^{n-1} \beta_k f_{i-k} + O(h^{n+1}) & \begin{array}{l} \text{open predictor} \\ \text{Adams - Bashforth} \end{array} \\ h \sum_{k=0}^{n-1} \bar{\beta}_k f_{i-k+1} + O(h^{n+1}) & \begin{array}{l} \text{closed corrector} \\ \text{Adams - Moulton} \end{array} \end{cases}$$

with  $\beta_k$  and  $\bar{\beta}_k$  determined from Tables 26.1 and 26.2, respectively.

## Popular Multi-Step Methods

1. Heun non-self starting — Newton-Cotes with  $n=1$
2. Milne's — Newton-Cotes with  $n=3$  (but use Hamming's corrector for better stability properties)
3. 4th-Order Adams
  - *Predictor 4th-Order Adams-Bashforth*
  - *Corrector 4th-Order Adams-Moulton*

*If predictor and corrector are of same order, we can obtain estimates of the truncation error during computation.*

## Improving Accuracy and Efficiency (use of modifiers)

1. provide criterion for step size adjustment (Adaptive Methods)
2. employ modifiers determined from error analysis

For Non-self-Starting (NSS) Heun:

- *Final Corrector Modifier*

$$E_c = -\frac{1}{5} \left( y_{i+1}^m - \hat{y}_{i+1}^0 \right) \quad y_{i+1}^m \leftarrow y_{i+1}^m - \frac{y_{i+1}^m - \hat{y}_{i+1}^0}{5}$$

- *Predictor Modifier (excluding 1<sup>st</sup> step)*

$$E_p = \frac{4}{5} \left( y_i^m - \hat{y}_i^0 \right) \quad \hat{y}_{i+1}^0 \leftarrow \hat{y}_{i+1}^0 + \frac{4}{5} \left( y_i^m - \hat{y}_i^0 \right)$$

with  $\hat{y}_i^0$  = unmodified  $\hat{y}_{i+1}^0$  from previous step

$y_i^m$  = unmodified  $y_{i+1}^m$  from previous step

## General Predictor-Corrector Schemes

*Errors cited are local errors. Global errors order  $h$  smaller.*

### Predictors:

*Euler:*

$$\hat{y}_{i+1} = y_i + hf(x_i, y_i) + O(h^2)$$

*Midpoint (same as Non-Self-Starting Heun):*

$$\hat{y}_{i+1} = y_{i-1} + 2hf(x_i, y_i) + O(h^3)$$

*Adams-Bashforth 2nd-Order:*

$$\hat{y}_{i+1} = y_i + \frac{h}{2} \{3f(x_i, y_i) - f(x_{i-1}, y_{i-1})\} + O(h^3)$$

*Adams-Bashforth 4th-Order:*

$$\hat{y}_{i+1} = y_i + \frac{h}{24} \{55f(x_i, y_i) - 59f(x_{i-1}, y_{i-1}) + 37f(x_{i-2}, y_{i-2}) - 9f(x_{i-3}, y_{i-3})\} + O(h^5)$$

*Hamming (and Milne's Method):*

$$\hat{y}_{i+1} = y_{i-3} + \frac{4h}{3} \{2f(x_i, y_i) - f(x_{i-1}, y_{i-1}) + f(x_{i-2}, y_{i-2})\} + O(h^5)$$



## Correctors:

- refine value of  $y_{i+1}$  given a  $\hat{y}_{i+1}$
- more stable numerically – no spurious solutions which go out of control

***Adams-Moulton 2nd-Order closed (Non-self starting Heun):***

$$y_{i+1} = y_i + h/2 \{f(x_{i+1}, \hat{y}_{i+1}) + f(x_i, y_i)\} + O(h^3)$$

***Adams-Moulton 4th-Order closed (relatively stable):***

$$y_{i+1} = y_i + h/24 \{9f(x_{i+1}, \hat{y}_{i+1}) - 19f(x_i, y_i) - 5f(x_{i-1}, y_{i-1}) + f(x_{i-2}, y_{i-2})\} + O(h^5)$$

***Milne's (N.C. 3-point, Simpson's 1/3 Rule, not always stable!!!):***

$$y_{i+1} = y_{i-1} + h/3 \{f(x_{i+1}, \hat{y}_{i+1}) + 4f(x_i, y_i) + f(x_{i-1}, y_{i-1})\} + O(h^5)$$

***Hamming (modified Milne, stable):***

$$y_{i+1} = 9/8 y_i - 1/8 y_{i-2} + 3h/8 \{f(x_{i+1}, \hat{y}_{i+1}) + 2f(x_i, y_i) + f(x_{i-1}, y_{i-1})\} + O(h^5)$$

## *Predictor-Corrector solution*

### **Example: Undamped, harmonically driven oscillator**

$$m \frac{d^2 x}{dt^2} + kx = P(t)$$

$m$  = mass

$k$  = spring constant

$P(t)$  = forcing function =  $P \sin(\gamma t)$

$\gamma$  = driving frequency

$$\omega = \sqrt{\frac{k}{m}} = \text{circular frequency}$$

$$T = \frac{2\pi}{\omega} = \text{natural period}$$

Initial conditions:  $x(0) = 0$  and  $\frac{dx}{dt}(0) = 0$

Analytical solution:

$$x(t) = \frac{-P\gamma}{\omega m(\omega^2 - \gamma^2)} \sin(\omega t) + \frac{P}{m(\omega^2 - \gamma^2)} \sin(\gamma t)$$

***Predictor-Corrector solution: Undamped, harmonic oscillator***

$$m \frac{d^2 x}{dt^2} + kx = P(t)$$

Recast 2nd-order ODE as two 1st-order ODE's:

$$\frac{dx}{dt} = v = f(t, x, v) \quad \text{with } x(0) = 0$$

$$\frac{dv}{dt} = \frac{P \sin(\gamma t)}{m} - \omega^2 x = g(t, x, v) \quad \text{with } v(0) = 0$$

Let  $m = 0.25$ ,  $k = 4\pi^2$ ,  $P = 20$ ,  $\gamma = 8$ ,

$$\text{thus: } \mathbf{g = 80 \sin(8t) - 16\pi^2 x}$$

Use 4th-ord. Adams Method to solve from  $t=0$  until  $t=1$  w/  $h=0.1$

***Predictor-Corrector solution:*** Undamped, harmonic oscillator

$$m \frac{d^2 x}{dt^2} + kx = P(t)$$

**First**, we need start-up values. Use 4th-order RK to get 4 points:

t	x(t)	y(t)
0.0	0.0000	0.0000
0.1	0.1038	2.6234
0.2	0.5324	5.1401
0.3	0.8692	0.4949

## ***Predictor-Corrector solution:* Undamped, harmonic oscillator**

**Second**, the Adams-Bashforth 4th-order predictor is:

$$y_{i+1} = y_i + \frac{h}{24} \{55 f(x_i, y_i) - 59 f(x_{i-1}, y_{i-1}) + 37 f(x_{i-2}, y_{i-2}) - 9 f(x_{i-3}, y_{i-3})\} + O(h^5)$$

**Example :**

We apply this predictor to both  $x$  and  $v$ :

$$\begin{aligned} x(0.4) &= x(0.3) + [55(0.4949) - 59(5.1401) + 37(2.6234) - 9(0.0)] = \\ &= \mathbf{0.1235} \end{aligned}$$

$$\begin{aligned} v(0.4) &= v(0.3) + (0.1/24) [55 g(0.3, 0.8692, 0.4949) \\ &\quad - 59 g(0.2, 0.5324, 5.1401) \\ &\quad + 37 g(0.1, 0.1038, 2.6234) - 9 g(0,0,0)] = \mathbf{-11.2492} \end{aligned}$$

$$\text{where } dv/dt = g(t, x, v) = 80 \sin 8t - 16\pi^2 x$$

## ***Predictor-Corrector solution:* Undamped, harmonic oscillator**

**Third**, the Adams-Moulton 4th-order corrector is:

$$y_{i+1} = y_i + \frac{h}{24} \{9f(x_{i+1}, y_{i+1}) + 19f(x_i, y_i) - 5f(x_{i-1}, y_{i-1}) + f(x_{i-2}, y_{i-2})\} + O(h^5)$$

We apply the corrector to **both** x and v to obtain the 1<sup>st</sup> iteration:

$$\begin{aligned} x(0.4) = x(0.3) + (0.1/24) [9 (-\mathbf{11.2492}) + 19(0.4949) \\ - 5(5.1401) + (2.6234)] = 0.8169 \end{aligned}$$

$$\begin{aligned} v(0.4) = v(0.3) + (0.1/24) [9 g(0.4, \mathbf{0.1235}, \mathbf{-11.2492}) \\ + 19 g(0.3, \dots) \dots] = -6.7435 \end{aligned}$$

$$\text{where } dv/dt = g(t, x, v) = 80 \sin 8t - 16\pi^2 x$$

We then can **iterate** until convergence:

$$x(0.4) = 0.8169, 0.842862, 0.843847, 0.843834, \dots$$

$$v(0.4) = -6.7435, -10.84973, -11.00371, -11.00949, \dots$$

**Do the Second and Third tasks for each successive step.**

## *Numerical Methods for ODE's*

### **Advantages of Runge-Kutta [such as $O(h^4)$ formula]**

- simple to apply
- self-starting (single-step)
- easy to change step size
- always stable (all  $h$ )
- can use matched pairs of different order to estimate local truncation error

### **Advantages of Predictor-Correctors**

#### **[such as Adams-Moulton Formula of $O(h^4)$ ]**

- without iteration is twice as efficient as Runge-Kutta 4th-Order
- local truncation error is easily estimated from difference so we can adjust step sizes and apply modifiers

For same step sizes, error terms are reasonably similar.

## *Summary*

### Single-step

Euler

Heun

Mid-point

RK4

### Multi-step (predictor-corrector)

Non-self-starting Heun

Milne (Newton-Cotes  $n=3$ )

Adams (Adams-Bashford, Adams-Moulton)

### Stiffness - stability

### Adaptive methods – local error estimates – modifiers

Half step

RK Fehlberg



## *Outline of our Study of ODE's*

### **I. Single-Step Methods for I.V. Problems (C&C Ch. 25)**

- a. Euler
- b. Heun and Improved Polygon
- c. General Runge-Kutta
- d. Adaptive step-size control

### **II. Stiff ODEs (C&C 26.1)**

### **III. Multi-Step Methods for I.V. Problems (C&C 26.2)**

- a. Non-Self-Starting Heun
- b. Newton-Cotes
- c. Adams

### **IV. Boundary Value Problems (C&C Ch. 27.1 & 27.2)**

- a. **Solution Methods**
  - 1. **Shooting Method**
  - 2. **Finite Difference Method**
- b. Eigenvalue Problems  $\{ [A] - \lambda [I] \} \{x\} = \{0\}$

## *ODE's – Boundary Value Problems*

**Recall:** Unique solution to an nth-order ODE  
requires n given conditions

**Conditions for a 2nd-order ODE (requires 2 given conditions):**

*Initial value problem*

$$y(x_0) = y_0$$
$$y'(x_0) = y'_0$$

*Boundary value problem*

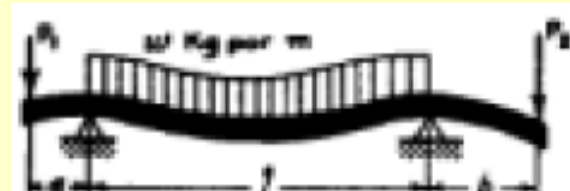
$$y(x_0) = y_0$$
$$y(x_f) = y_f$$



$$y(0) = y'(0) = 0, y''(l) = y'''(l) = 0.$$

***Beam Equation:***

$$EI y^{IV}(x) = -p(x)$$



$$y(0) = y(l) = 0, y''(0) = y''(l) = 0$$

## *Beam Equation*

### **Sailboat mast deflection problem**

$$\frac{d^4 v}{dz^4} = \frac{f(z)}{EI} \quad \text{(Euler-Bernoulli Law of Bending)}$$

**at the base of mast:**  $v(0) = 0$ ;  $v'(0) = 0$ ;

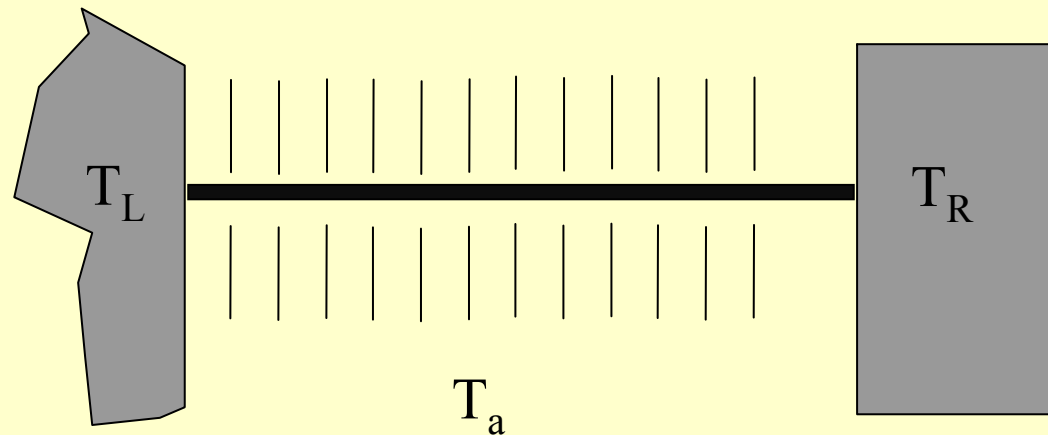
**at the top of mast:**  $v''(L) = 0$ ;  $v'''(L) = 0$

where:

- $E$  = Modulus of Elasticity (material property)
- $I$  = Moment of Inertia (geometry of material)
- $f(z)$  = wind pressure at height  $z$
- $v(x)$  = deflection (displacement) from vertical
- $z$  = height
- $E I v''$  = moment
- $E I v'''$  = shear (applied force)

*ODE's – Boundary Value Problem* (C&C 27.1)

$$\frac{d^2T}{dx^2} = h'(T - T_a)$$



Solve for the steady-state  
temperature distribution in the rod  
For a given  $T_L$  and  $T_R < T_L$

## *ODE's – Boundary Value Problem – Shooting Method*

$$\text{Solve } = \frac{d^2y}{dx^2} = f(x, y, y') \quad \text{given } y(x_0) = y_0 \text{ and } y(x_f) = y_f$$

1. Convert 2nd-Order ODE to two 1st-Order ODE's

$$\frac{dy}{dx} = z \qquad \frac{dz}{dx} = f(x, y, z)$$

2. Given initial condition,  $y(x_0) = y_0$ ,  
estimate (best guess) the initial condition  $z(x_0) = z_0^{(1)}$
3. Solve ODE using the assumed initial values from  $x = x_0$  to  $x = x_f$   
using stepping methods. Because we **estimated** the initial  
conditions for  $z(x_0)$ ; will find  $y(x_f) \neq y_f$

## *ODE's – Boundary Value Problem – Shooting Method*

4. Re-estimate initial condition  $z(x_0) = z_0^{(2)}$ , and again solve the assumed initial value ODE from  $x = x_0$  to  $x = x_f$ .

Because we estimated the I.C.  $z(x_0)$ ,  $y(x_f) \neq y_f$ .

5. Interpolate or extrapolate to find the "correct"  $z(x_0) = z_0$

Given  $y_f$ ,  $(y_f^{(1)}, z_0^{(1)})$ , and  $(y_f^{(2)}, z_0^{(2)})$ :

$$z_0 = z_0^{(1)} + \frac{z_0^{(2)} - z_0^{(1)}}{y_f^{(2)} - y_f^{(1)}} (y_f - y_f^{(1)})$$

[This equation  
is not in C&C  
in general form]

**If ODE is linear**,  $z_0$  is the correct solution

**If ODE is nonlinear**, iterate until  $y_f^{(h)} \approx y_f$

## *Boundary Value Problem: Classical Shooting Method*

**Example: Single-degree-of-freedom, undamped, harmonically driven oscillator (see also C&C 28.4, p. 797)**

ODE is 2nd order:

$$m \frac{d^2 x}{dt^2} + kx = P(t) \quad \text{or} \quad \frac{d^2 x}{dt^2} + \omega^2 x = \frac{P(t)}{m}$$

$m$  = mass

$k$  = spring constant

$P(t)$  = forcing function =  $P \sin(\gamma t)$

$\omega = \sqrt{\frac{k}{m}}$  = circular frequency

$T = \frac{2\pi}{\omega}$  = natural period

Now, we are given **boundary** conditions:

$$x(0) = 0 \quad \text{and} \quad x(1) = 0.5$$

(that is, the initial condition  $v = dx/dt$  at  $t = 0$  is unknown,  $v_0$  )

## *Boundary Value Problem: Classical Shooting Method*

**Use the Shooting Method by employing a 4<sup>th</sup>-order RK**

$$\frac{d^2x}{dt^2} + \omega^2 x = \frac{P(t)}{m}$$

Recast the 2<sup>nd</sup>-order ODE as two 1<sup>st</sup>-order ODEs:

$$\frac{dx}{dt} = v = f(t, x, v) \quad \text{with } x(0) = 0$$

$$\frac{dv}{dt} = \frac{P \sin(\gamma t)}{m} - \omega^2 x = g(t, x, v) \quad \text{with } v(0) = v_0$$

Let  $m = 0.25$ ,  $k = 4\pi^2$ ,  $P = 20$ ,  $\gamma = 8$ ,

and thus  $g(t, x, v) = 80 \sin(8t) - 16\pi^2 x$ .



## *Boundary Value Problem: Classical Shooting Method*

Use 4th-ord. RK Method to solve from  $t=0$  until  $t=1$  w/  $h=0.1$ .

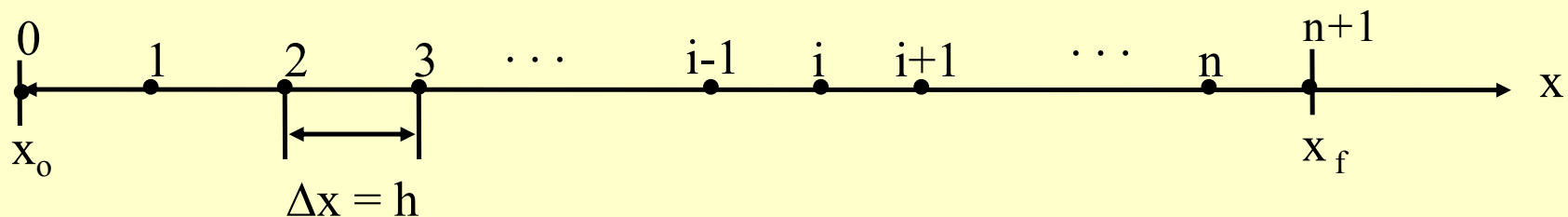
1. Guess  $v(0) = 1.0$
2. Find  $x(1) = \mathbf{0.91687}$  (Note: not equal to 0.5)
3. Guess  $v(0) = 100$
4. Find  $x(1) = \mathbf{0.09332}$
5. Because our ODE is linear, we can always interpolate  $\rightarrow$  yields the required initial value.

$$v(0) = \frac{100 - 1.0}{0.09332 - 0.91687} (0.5 - 0.91687) = \mathbf{51.113}$$

6. With  $v(0) = 51.113$ , check for  $x(1) = 0.5000$       **OK**

## *Finite Difference Method*

- The shooting method is inefficient for higher-order equations with several boundary conditions.
- Finite Difference method has advantage of being direct (not iterative) for linear problems, but requires the solution of simultaneous algebraic equations.



## *Finite Difference Method*

### **Approach:**

1. Divide domain to obtain  $n$  interior discrete points (usually evenly spaced @  $h$ ).
2. Write a finite difference expression for the ODE at each interior point.
3. Use known values of  $y$  at  $x = x_0$  and  $x = x_f$
4. Set up  $n$  linear equations with  $n$  unknowns. System is banded and often symmetric, so solve with an efficient method.

**Note:** If higher-order FDD equations and/or centered differences are used, you may need to employ imaginary or "phantom" points outside of domain to express B.C.'s (together with appropriate FD versions of B.C.'s).

## *Finite Difference Method*

### **Sailboat mast deflection problem**

$$\frac{d^4 v}{dz^4} = \frac{f(z)}{EI} \quad \text{(Euler-Bernoulli Law of Bending)}$$

**at the base of mast:**  $v(0) = 0$ ;  $v'(0) = 0$ ;

**at the top of mast:**  $v''(L) = 0$ ;  $v'''(L) = 0$

where:

- $E$  = Modulus of Elasticity (material property)
- $I$  = Moment of Inertia (geometry of material)
- $f(z)$  = wind pressure at height  $z$
- $v(x)$  = deflection (displacement) from vertical
- $z$  = height
- $E I v''$  = moment
- $E I v'''$  = shear (applied force)

## *Finite Difference Method*

**Sailboat mast deflection problem:**  $\frac{d^4 v}{dz^4} = v'''' = \frac{f(z)}{EI}$

at the base ( $z = 0$ ):  $v(0) = 0$ ;  $v'(0) = 0$ ;

at the top ( $z = L$ ):  $v''(L) = 0$ ;  $v'''(L) = 0$

$$v'''' = \frac{d^4 v}{dx^4} \approx \frac{v_{i-2} - 4v_{i-1} + 6v_i - 4v_{i+1} + v_{i+2}}{h^4} \quad [\text{dist. load, } f(z)]/EI$$

$$v''' = \frac{d^3 v}{dx^3} \approx \frac{-v_{i-2} + 2v_{i-1} - 2v_{i+1} + v_{i+2}}{2h^3} \quad (\text{shear})/EI$$

$$v'' = \frac{d^2 v}{dx^2} \approx \frac{v_{i-1} - 2v_i + v_{i+1}}{h^2} \quad (\text{bending moment})/EI$$

$$v' = \frac{dv}{dx} \approx \frac{v_{i-1} - v_{i+1}}{2h} \quad (\text{slope})$$

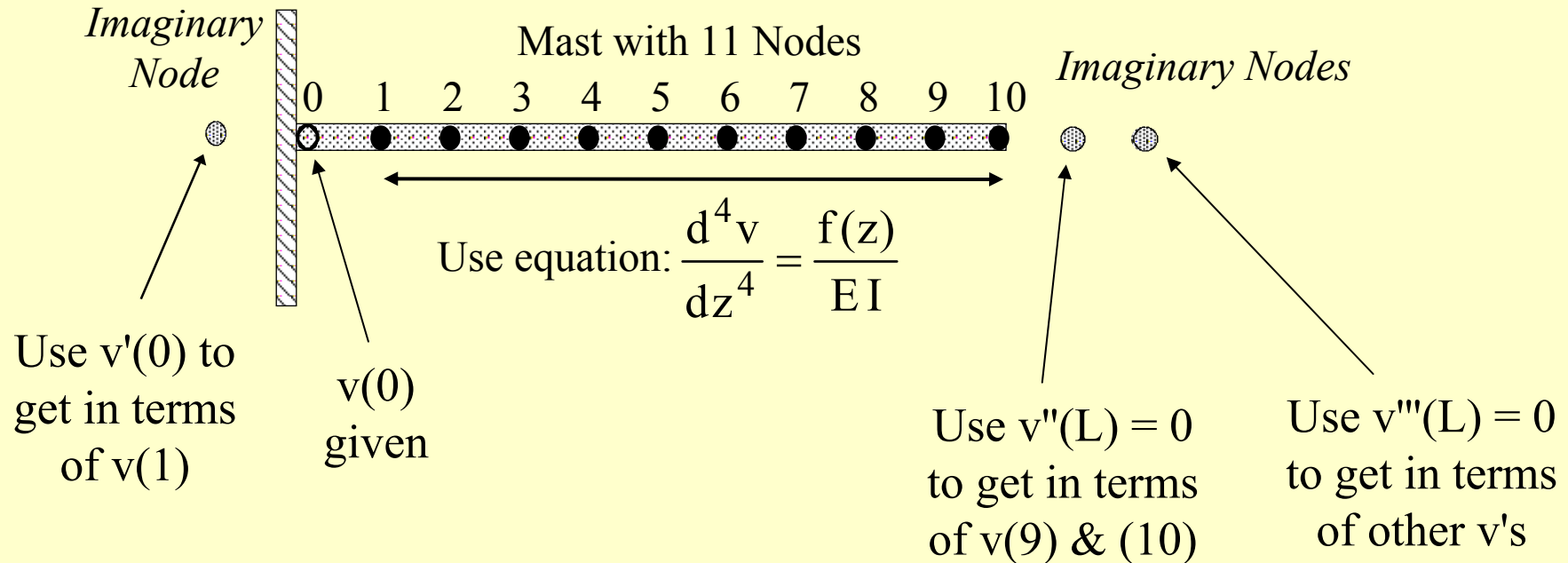
# Finite Difference Method

**Sailboat mast deflection problem:**

$$\frac{d^4 v}{dz^4} = \frac{f(z)}{EI}$$

**at the base:**  $v(0) = 0$ ;  $v'(0) = 0$ ;

**at the top:**  $v''(L) = 0$ ;  $v'''(L) = 0$



## *Finite Difference Method*

$$\frac{d^4 v}{dz^4} = \frac{f(z)}{EI}$$

@ pt.  $i$ :  $v_{i-2} - 4 v_{i-1} + 6 v_i - 4 v_{i+1} + v_{i+2} = h^4 f(x_i)/EI$

Using  $h = L/10$ , write the FD equations at points 1, 2, 9 and 10:

@  $i = 1$   $v_{-1} - 4 v_0 + 6 v_1 - 4 v_2 + v_3 = h^4 f(x_1)/EI$  (1)

@  $i = 2$   $v_0 - 4 v_1 + 6 v_2 - 4 v_3 + v_4 = h^4 f(x_2)/EI$  (2)

@  $i = 9$   $v_7 - 4 v_8 + 6 v_9 - 4 v_{10} + v_{11} = h^4 f(x_9)/EI$  (3)

@  $i = 10$   $v_8 - 4 v_9 + 6 v_{10} - 4 v_{11} + v_{12} = h^4 f(x_{10})/EI$  (4)

We now need to eliminate  $v_{-1}$ ,  $v_0$ ,  $v_{11}$ , and  $v_{12}$   
with the four boundary conditions

## *Finite Difference Method*

$$\frac{d^4 v}{dz^4} = \frac{f(z)}{EI}$$

Eliminating  $v_{-1}$ ,  $v_0$ ,  $v_{11}$ , and  $v_{12}$  with the four Boundary conditions

$$v(0) = 0:$$

$$v_0 = 0$$

$$v'(0) = 0:$$

$$\frac{v_{-1} - v_1}{2h} = 0$$

$$v_{-1} = v_1$$

$$v''(L) = 0:$$

$$\frac{v_9 - 2v_{10} + v_{11}}{h^2} = 0$$

$$v_{11} = 2v_{10} + v_9$$

$$v'''(L) = 0:$$

$$\frac{-v_8 + 2v_9 - 2v_{11} + v_{12}}{2h^3} = 0$$

$$v_{12} = v_8 - 4v_9 + 4v_{10}$$



## *Finite Difference Method*

$$\frac{d^4 v}{dz^4} = \frac{f(z)}{EI}$$

We substitute these into equations (1) to (4) to eliminate the unknowns at the “imaginary” points

$$v_0 = 0$$

$$v_{-1} = v_1$$

$$v_{11} = 2v_{10} + v_9$$

$$v_{12} = v_8 - 4v_9 + 4v_{10}$$

$$① i = 1 \quad 7 v_1 - 4 v_2 + v_3 = h^4 f(x_1) / EI \quad (1a)$$

$$① i = 2 \quad -4 v_1 + 6 v_2 - 4 v_3 + v_4 = h^4 f(x_2) / EI \quad (2a)$$

$$① i = 9 \quad v_7 - 4 v_8 + 5 v_9 - 2 v_{10} = h^4 f(x_9) / EI \quad (3a)$$

$$① i = 10 \quad v_8 - 4 v_9 + 2 v_{10} = h^4 f(x_{10}) / EI \quad (4a)$$

## *Finite Difference Method*

$$\begin{bmatrix}
 7 & -4 & 1 & & & & & & & & \\
 -4 & 6 & -4 & 1 & & & & & & & \\
 1 & -4 & 6 & -4 & 1 & & & & & & \\
 & 1 & -4 & 6 & -4 & 1 & & & & & \\
 & & 1 & -4 & 6 & -4 & 1 & & & & \\
 & & & 1 & -4 & 6 & -4 & 1 & & & \\
 & & & & 1 & -4 & 6 & -4 & 1 & & \\
 & & & & & 1 & -4 & 6 & -4 & 1 & \\
 & & & & & & 1 & -4 & 5 & -2 & \\
 & & & & & & & 1 & -4 & 2 & 
 \end{bmatrix}
 \begin{Bmatrix}
 v_1 \\
 v_2 \\
 v_3 \\
 v_4 \\
 v_5 \\
 v_6 \\
 v_7 \\
 v_8 \\
 v_9 \\
 v_{10}
 \end{Bmatrix}
 = \frac{h^4 f}{EI}
 \begin{Bmatrix}
 f(x_1) \\
 f(x_2) \\
 f(x_3) \\
 f(x_4) \\
 f(x_5) \\
 f(x_6) \\
 f(x_7) \\
 f(x_8) \\
 f(x_9) \\
 f(x_{10})
 \end{Bmatrix}$$

## *Finite Difference Method*

Once we have solved for all the  $v_i$ , we can obtain secondary results such as bending moments and shear forces by substituting the finite-divided-difference operators and the values of the  $v_i$  into such equations as:

$$M = EI v''$$

$$V = EI v'''$$

For more refined results, we can use a smaller  $h$  and more segments.

## *Merits of Different Numerical Methods for ODE Boundary Value Problems*

- **Shooting method**

Conceptually simple and easy.

Inefficient for higher-order systems w/ many boundary conditions.

May not converge for nonlinear problems.

Can blow up for bad guess of initial conditions.

- **Finite Difference method**

Stable

Direct (not iterative) for linear problems.

Requires solution of simultaneous algebraic eqns.

More complex.

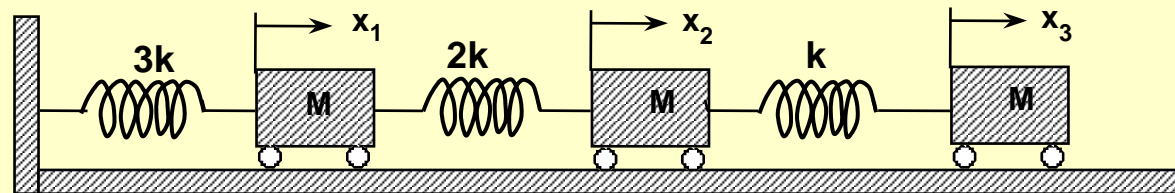
*FD better suited for eigenvalue problems.*

## Engineering Applications of Eigenvalue Problems

Natural periods and modes of vibration of structures and dynamical systems.  
(C&C Ex. 27.4, p. 761)

**Application:** determination of the natural frequencies of a system

**Example:** A mass-spring system with three identical (frictionless) masses connected by three springs with different spring constants.



The displacement of each spring is measured relative to its own local coordinate system with an origin at the spring's equilibrium position.

$$m \frac{d^2 x_1}{dt^2} = -3kx_1 - 2k(x_1 - x_2)$$

$$m \frac{d^2 x_2}{dt^2} = -2k(x_2 - x_1) - k(x_2 - x_3)$$

$$m \frac{d^2 x_3}{dt^2} = -k(x_3 - x_2)$$

## *Engineering Applications of Eigenvalue Problems*

If one assumes that  $x$  has the form:

$$x_i = a_i \cos \omega t ; \quad d^2x/dt^2 = -\omega^2 a_i \cos \omega t$$

Then, with  $\lambda = m \omega^2 / k$ , the governing equations become:

$$-\frac{m\omega^2}{k} a_1 = -5a_1 + 2a_2 \quad \implies \quad 5a_1 - 2a_2 = \lambda a_1$$

$$-\frac{m\omega^2}{k} a_2 = 2a_1 - 3a_2 + a_3 \quad \implies \quad -2a_1 + 3a_2 - a_3 = \lambda a_2$$

$$-\frac{m\omega^2}{k} a_3 = a_2 - a_3 \quad \implies \quad -a_2 + a_3 = \lambda a_3$$

## *Engineering Applications of Eigenvalue Problems*

$$\begin{aligned} 5a_1 - 2a_2 &= \lambda a_1 \\ -2a_1 + 3a_2 - a_3 &= \lambda a_2 \\ -a_1 + a_2 &= \lambda a_3 \end{aligned} \implies \begin{bmatrix} 5 & -2 & 0 \\ -2 & 3 & -1 \\ 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} a_1 \\ a_2 \\ a_3 \end{Bmatrix} = \lambda \begin{Bmatrix} a_1 \\ a_2 \\ a_3 \end{Bmatrix}$$

$[\mathbf{A}] \quad \{\mathbf{x}\} = \lambda \quad \{\mathbf{x}\}$

$$\text{or} \quad \begin{bmatrix} 5 - \lambda & -2 & 0 \\ -2 & 3 - \lambda & -1 \\ 0 & -1 & 1 - \lambda \end{bmatrix} \begin{Bmatrix} a_1 \\ a_2 \\ a_3 \end{Bmatrix} = \mathbf{0}$$

$[\mathbf{A} - \mathbf{I} \lambda] \quad \{\mathbf{x}\} = \mathbf{0}$

## *Engineering Applications of Eigenvalue Problems*

For a non-trivial solution:

$$\det [\mathbf{A} - \mathbf{I} \lambda] = 0 \quad \rightarrow \quad \text{find} \quad \det \begin{bmatrix} 5 - \lambda & -2 & 0 \\ -2 & 3 - \lambda & -1 \\ 0 & -1 & 1 - \lambda \end{bmatrix} = 0$$

The determinant yields a cubic equation for  $\lambda$ :

$$(5 - \lambda) [(3 - \lambda)(1 - \lambda) - (-1)(-1)] - 2 [(-1)(0) - (-2)(1 - \lambda)] = 0$$

$$(5 - \lambda) [2 - 4\lambda + \lambda^2] - 2 [2 - 2\lambda] = 0$$

$$6 - 18\lambda + 9\lambda^2 - \lambda^3 = 0$$

The three solutions of the cubic equation are the three eigenvalues:

$$\lambda_1 = 6.29$$

$$\lambda_2 = 2.29$$

$$\lambda_3 = 0.42$$

Fast oscillation

Slow oscillation



## *Engineering Applications of Eigenvalue Problems*

The corresponding eigenvectors  $\{a_i\}$  are found by solving:

$$\begin{bmatrix} 5 - \lambda_i & -2 & 0 \\ -2 & 3 - \lambda_i & -1 \\ 0 & -1 & 1 - \lambda_i \end{bmatrix} \begin{Bmatrix} a_{1i} \\ a_{2i} \\ a_{3i} \end{Bmatrix} = 0$$

Because this is a homogeneous equation, we can only find the relative values of the  $a_i$ 's. For  $\lambda_1 = 6.29$

$$\begin{bmatrix} 5 - 6.29 & -2 & 0 \\ -2 & 3 - 6.29 & -1 \\ 0 & -1 & 1 - 6.29 \end{bmatrix} \begin{Bmatrix} a_{1i} \\ a_{2i} \\ a_{3i} \end{Bmatrix} = 0$$

$$\frac{a_2}{a_1} = -0.645 \quad \frac{a_3}{a_1} = 0.122$$

## *Engineering Applications of Eigenvalue Problems*

$$\frac{a_2}{a_1} = -0.645 \quad \frac{a_3}{a_1} = 0.122$$

Only the relative values of the  $a_i$ 's are significant. Thus setting  $a_1 = 1.00$  we have:

**Fast**

for  $\lambda = 6.29$

$$\{a\}_1 = \begin{Bmatrix} 1.000 \\ -0.645 \\ 0.122 \end{Bmatrix}$$

for  $\lambda_2 = 2.29$

$$\{a\}_2 = \begin{Bmatrix} 0.74 \\ 1.000 \\ -0.77 \end{Bmatrix}$$

**Slow**

for  $\lambda_2 = 0.42$

$$\{a\}_2 = \begin{Bmatrix} 0.25 \\ 0.55 \\ 1.000 \end{Bmatrix}$$

## *Engineering Applications of Eigenvalue Problems*

Thus, there are **3 possible natural frequencies**.

Recall that the frequency is defined as  $\omega = \sqrt{\frac{\lambda k}{m}}$

$$[A] \{x\} = \lambda \{x\}$$

$$[A - I \lambda] \{x\} = 0$$

$$\det [A - I \lambda] = 0 \implies \text{find } \lambda$$

- If  $[A]$  is  $n \times n$ , there are  $n$  eigenvalues  $\lambda_i$   
and  $n$  eigenvectors  $\{x\}_i$
- If  $[A]$  is symmetric, the eigenvectors are orthogonal:

$$\begin{aligned} \{x\}_i^T \{x\}_j &= 0 && \text{if } j \neq i \\ &= 1 && \text{if } j = i \end{aligned}$$

## ***POWER METHOD for Eigenvector Analysis***

(An iterative approach for solving for eigenvalues & eigenvectors)

Assume that a vector  $\{y\}$  can be expressed as a linear combination of the eigenvectors:

$$\{y\} = b_1\{x_1\} + b_2\{x_2\} + \dots + b_n\{x_n\} = \sum_{i=1}^n b_i\{x_i\}$$

Multiplying the above equation by  $[A]$  yields:

$$[A]\{y\} = \sum_{i=1}^n b_i[A]\{x_i\} = \sum_{i=1}^n b_i\lambda_i\{x_i\}$$

Ordering the  $\lambda_i$  so that  $|\lambda_1| > |\lambda_2| > |\lambda_3| > \dots > |\lambda_n|$

$$\text{Then } [A]\{y\} = \lambda_1 \left( b_1\{x_1\} + \sum_{i=2}^n b_i \left( \frac{\lambda_i}{\lambda_1} \right) \{x_i\} \right) \text{ where } \left| \frac{\lambda_i}{\lambda_1} \right| < 1$$

## ***POWER METHOD for Eigenvector Analysis***

Multiplying the equation by  $[A]$  again, we have:

$$[A]^2 \{y\} = \lambda_1^2 \left( b_1 \{x_1\} + \sum_{i=2}^n b_i \left( \frac{\lambda_i}{\lambda_1} \right)^2 \{x_i\} \right)$$

Repeating this process  $m$  times:

$$[A]^m \{y\} = \lambda_1^m \left( b_1 \{x_1\} + \sum_{i=2}^n b_i \left( \frac{\lambda_i}{\lambda_1} \right)^m \{x_i\} \right)$$

Since  $\left( \frac{\lambda_i}{\lambda_1} \right) \Rightarrow 0$  for all  $i \neq 1$  as  $m \Rightarrow \infty$ ,

$$\text{then } [A]^m \{y\} \rightarrow \lambda_1^m b_1 \{x_1\}$$

Since we can only know the relative values of the elements of  $\{x\}$ , we may normalize  $\{x\}$ . If  $\{x\}$  is normalized such that the largest element is equal to  $\{1\}$ , then is the first eigenvalue,  $\lambda_1$

## *POWER METHOD Numerical Example*

$$\begin{bmatrix} 3 & 7 & 9 \\ 9 & 4 & 3 \\ 9 & 3 & 8 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \lambda \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}$$

Find the maximum eigenvector. Initial guess:  $\{y\}^T = \{1 \ 1 \ 1\}$

$$[A] \{y\} = \begin{bmatrix} 3 & 7 & 9 \\ 9 & 4 & 3 \\ 9 & 3 & 8 \end{bmatrix} \begin{Bmatrix} 1 \\ 1 \\ 1 \end{Bmatrix} = \begin{Bmatrix} 19 \\ 16 \\ 20 \end{Bmatrix} \implies 20 \quad \begin{Bmatrix} 0.95 \\ 0.80 \\ 1.00 \end{Bmatrix}$$

Approx. eigenvalue      eigen-vector

## ***POWER METHOD Numerical Example***

**2<sup>nd</sup> iteration**

$$[A]^2 \{y\} = \begin{bmatrix} 3 & 7 & 9 \\ 9 & 4 & 3 \\ 9 & 3 & 8 \end{bmatrix} \begin{Bmatrix} 0.95 \\ 0.80 \\ 1.00 \end{Bmatrix} = 18.95 \begin{Bmatrix} 0.92084 \\ 0.77836 \\ 1.00 \end{Bmatrix}$$

Approximate error,  $\varepsilon_a$ :

$$\max_j \left| \frac{(e_{i,j} - e_{i-1,j})}{e_{i,j}} \right| * 100\% = \left| \frac{0.92084 - 0.95}{0.92084} \right| * 100\% = 4.2\%$$

**3<sup>rd</sup> iteration**

$$[A]^3 \{y\} = \begin{bmatrix} 3 & 7 & 9 \\ 9 & 4 & 3 \\ 9 & 3 & 8 \end{bmatrix} \begin{Bmatrix} 0.92084 \\ 0.77836 \\ 1.00 \end{Bmatrix} = 18.623 \begin{Bmatrix} 0.92420 \\ 0.77331 \\ 1.00 \end{Bmatrix}$$

$$\varepsilon_a = \left| \frac{0.92084 - 0.95}{0.92084} \right| * 100\% = 0.75\%$$

## *POWER METHOD Numerical Example*

**After 10 iterations:**

$$\lambda = 18.622 \quad \{\mathbf{x}\} = \begin{Bmatrix} 0.92251 \\ 0.77301 \\ 1.00 \end{Bmatrix} \quad \varepsilon_a = 0.00033\%$$



## *Engineering Applications of Eigenvalue Problems*

1. **Natural periods & modes of vibration** of dynamic systems and structures. Boundary-value problem from separation of variables:

$$m * a = \text{force} \Rightarrow m(x) \frac{\partial^2 y(x, t)}{\partial t^2} = k(x) \frac{\partial^2 y}{\partial x^2}$$

If one assumes  $y(x, t) = \exp(i\omega t) u(x)$

then displacement function  $u(x)$  satisfies ODE:

$$-\omega^2 u(x) = [k(x)/m(x)] d^2u/dx^2$$

which may be written:  $a(x) d^2u/dx^2 + \omega^2 u(x) = 0$

With a FD approximation of  $u_{,xx} = d^2u/dx^2$  this becomes:

$$[\mathbf{A}] \{\mathbf{u}_i\} = -\omega^2 \{\mathbf{u}_i\}$$

## *Engineering Applications of Eigenvalue Problems*

### **2. Buckling loads and modes of structures (C&C 27.2.3, p. 762)**

Deflection of vertically loaded beam with horizontally constrained end:

$$EI y'' - Py = 0; \quad y(0) = 0; \quad y(L) = 0.$$

Is there deflection?

3. Directions and values of principal stresses.
4. Directions and values of principal moments of inertia.
5. Condition numbers linear systems of equations.
6. Stability criteria for numerical solution of PDE's.
7. Other problems in which "principal values" are sought.

# *Engineering Applications of Eigenvalue Problems*

## **Matrix Form of Eigenvalue Problem**

$$[A] \{x\} = \lambda \{x\}$$

$$\rightarrow ([A] - \lambda [I]) \{x\} = 0 \quad \text{but want } \{x\} \neq 0$$

## **Computation of Eigenvalues and Eigenvectors**

1. Power and Inverse Power method for largest and smallest eigenvalues. (C&C 27.2.5, p. 767)
2. Direct numerical algorithms: Jacobi, Given, Householder, and QR factorization. (C&C 27.2.6, p. 770)
3. Use of characteristic polynomial for “toy” problems. (C&C 27.2.4, p. 765)

# *Engineering Applications of Eigenvalue Problems*

## **Power Method for the largest Eigenvalue**

Compute  $\mathbf{x}_{t+1} = \mathbf{A} \mathbf{x}_t / \|\mathbf{x}_t\|$

for larger and larger  $t$  to estimate largest  $\lambda(\mathbf{A})$ .

Always works for Symmetric  $\mathbf{A}$ .

## *Eigenvalue Problems: Power Method*

### **Inverse Power Method for the smallest Eigenvalue**

Compute  $\mathbf{x}_{t+1} = \mathbf{B} \mathbf{x}_t / \|\mathbf{x}_t\|$

for larger and larger  $t$  to estimate largest  $\lambda(\mathbf{B})$ .

Here  $\mathbf{B} = \mathbf{A}^{-1}$ . Eigenvalues of  $\mathbf{B}$  are inverse of those of  $\mathbf{A}$ .

**BUT WE DO NOT COMPUTE  $\mathbf{A}^{-1}$  ! (Don't do it.)**

Instead use LU decomposition of  $\mathbf{A}$ .

## *Engineering Applications of Eigenvalue Problems*

### **Shifting method for any Eigenvalue:**

Suppose we want to compute eigenvector whose eigenvalue is about  $\delta$ .

Let  $B = (A - \delta I)^{-1}$  and compute  $x_{t+1} = B x_t / \|x_t\|$

for larger and larger  $t$  to estimate largest  $\lambda[(A - \delta I)^{-1}]$ .

This will compute the eigenvalue *nearest* to  $\delta$  !

**But we DO NOT COMPUTE INVERSE.** Use LU decomposition.

If  $A$  has eigenvector-value pairs  $(e_i, \lambda_i)$ ,

$$(A - \delta I) e_i = A e_i - \delta I e_i = \lambda_i e_i - \delta e_i = (\lambda_i - \delta) e_i$$

Thus shifted matrix has same eigenvectors  $e_i$  as  $A$  with shifted eigenvalues  $(\lambda_i - \delta)$ .

Hence  $(A - \delta I)^{-1}$  has eigenvalues  $(\lambda_i - \delta)^{-1}$ .